

```
In [1]: from numpy import *
import matplotlib.pyplot as plt
%matplotlib inline
```

## Numerical Integration with E-P Behaviour

Here we are going to find the analytical solution of the 3rd problem of the homework #1 and next the numerical solution, using the constant acceleration algorithm.

## Problem Data

and some easily derived quantities.

```
In [2]: mass = 400
wn = 2*pi*4
zeta = 0.03
p0, t0 = 8200.0, 0.040
fy = 3200.0

stif = mass*wn**2
damp = 2*zeta*wn*mass
Tn = 2*pi/wn
beta = Tn/2/t0

Dst, xy = p0/stif, fy/stif
```

## Particular Integral

$$\xi(t) = C \sin(\beta\omega_n t) + D \cos(\beta\omega_n t)$$

```
In [3]: det = (1-beta**2)**2 + (2*zeta*beta)**2  
C = (1-beta**2)/det  
D = -2*zeta*beta/det
```

## Homogeneous Solution and Initial Conditions

$$x(t) = \exp(-\zeta\omega_n t)(A \sin(\delta\omega_n t) + B \cos(\delta\omega_n t)) + \xi(t), \quad \delta = \sqrt{1 - \zeta^2}.$$

$$x(0) = B + D = 0 \rightarrow B = -D.$$

$$v(0) = \omega_n(-\zeta B + \delta A + \beta C) = \omega_n(\zeta D + \delta A + \beta C) = 0 \rightarrow A = -(\zeta D + \beta C).$$

/  $\delta$

```
In [4]: d = sqrt(1-zeta**2)
B = -D
A = (-zeta*D-beta*C) / d
```

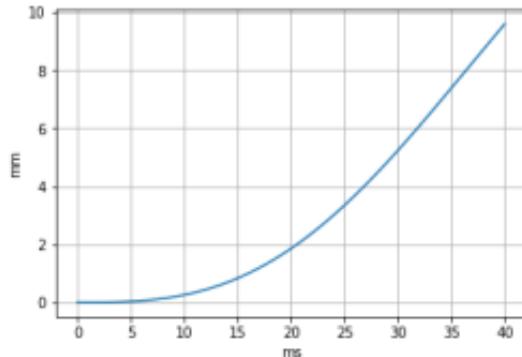
## Compute the forced response

For simplicity we use the adimensional time variable  $a = \omega_n t$ .

```
In [5]: z, b = zeta, beta # shorter aliases  
t = linspace(0, t0, 401)  
a = wn*t  
  
x = Dst*( exp(-z*a)*(A*sin(d*a)+B*cos(d*a)) + C*sin(b*a) + D*cos(b*a))  
v = (exp(-z*a) * ( -z*(A*sin(d*a)+B*cos(d*a)) +d*(A*cos(d*a)-B*sin(d*a))))  
v = Dst*wn*(v + b*(C*cos(b*a)-D*sin(b*a)))
```

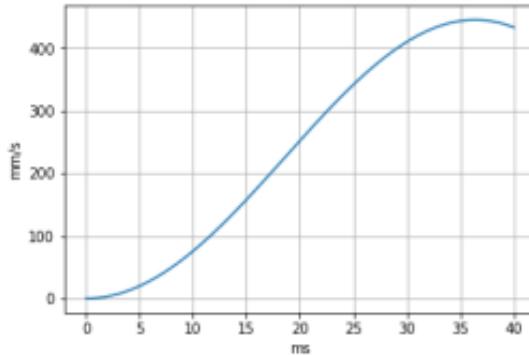
## Plot of Displacements

```
In [6]: plt.plot(1000*t, 1000*x); plt.grid()
plt.ylabel('mm') ; plt.xlabel('ms')
plt.show()
```



## Plot of Velocities

```
In [7]: plt.plot(1000*t, 1000*v); plt.grid()
plt.ylabel('mm/s') ; plt.xlabel('ms')
plt.show()
```



## Free Elastic Response

$$x(\tau) = \exp(-\zeta\omega_n\tau)(E \sin(\delta\omega_n\tau) + F \cos(\delta\omega_n\tau)), \quad \tau = t - t_0$$

$$x(0) = F = x(t_0)$$

$$v(0) = \omega_n(-\zeta F + \delta E) = v(t_0)$$

```
In [8]: x0, v0 = x[-1], v[-1] # the initial values of x, v are the last element  
F = x0  
E = (v0/wn+z*F)/d
```

## Evaluate the Elastic Response

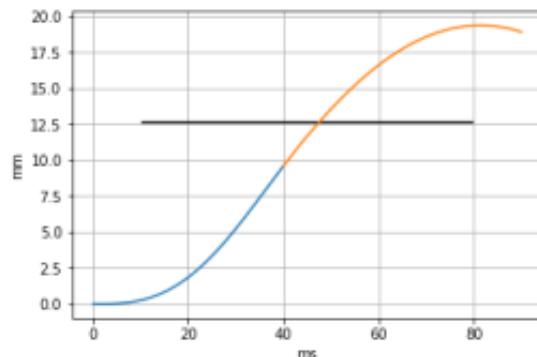
```
In [9]: t2 = linspace(0,0.05,501)
a2 = wn*t2
x2 = exp(-z*a2)*(E*sin(d*a2)+F*cos(d*a2))
v2 = wn*exp(-z*a2)*(-z*(E*sin(d*a2)+F*cos(d*a2)) + d*(E*cos(d*a2)-F*sin(d*a2)))
```

## Plot the Elastic Displacements

and an horizontal line indicating the value of  $x_y$ ...

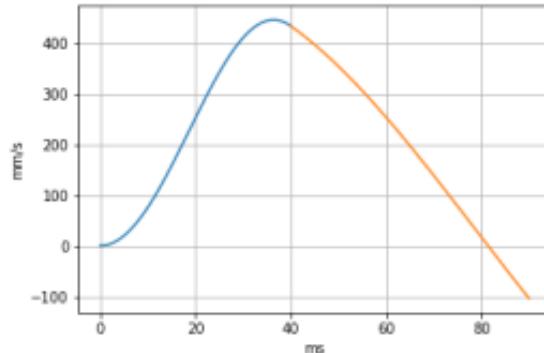
```
In [10]: plt.plot(1000*t, 1000*x); plt.grid()
plt.plot(1000*(t2+t0), 1000*x2)
plt.ylabel('mm') ; plt.xlabel('ms')
plt.hlines(xy*1000, 10, 80)
print('Max elastic response is %f mm.'%(1000*max(x2)))
```

Max elastic response is 19.361712 mm.



## Plot the Elastic Velocities

```
In [11]: plt.plot(1000*t, 1000*v); plt.grid()
plt.plot(1000*(t2+t0), 1000*v2)
plt.ylabel('mm/s') ; plt.xlabel('ms');
```



## **Plastic Response**

We need the elastic response, that we already obtained at discrete points of time, as functions of a time variable, so that we can find the instant of yielding (using a library function) and then, using the info on the initial velocity, find the integral of the plastic response.

## Elastic Response as Functions

```
In [12]: f0 = lambda s: exp(-z*wn*s)*(E*sin(d*wn*s)+F*cos(d*wn*s))
f1 = lambda s: wn*exp(-z*wn*s)*(-z*(E*sin(d*wn*s)+F*cos(d*wn*s)) +
d*(E*cos(d*wn*s)-F*sin(d*wn*s)))
```

## Find $t_y$ and the initial conditions for plastic branch

Now  $\tau$  is  $\tau = t - (t_0 + t_y)$ ...

```
In [13]: from scipy.optimize import newton
ty = newton(lambda s: f0(s)-xy, 0.01)

xp = f0(ty)
vp = f1(ty)
```

## Plastic Response

### Particular Integral

The external force is  $-f_y$ , it is  $\xi = S\tau$ , substituting we have  $cS = -f_y$  and  $S = -f_y/c$

```
In [14]: S = -fy/damp
```

### Hom. solution and initial conditions

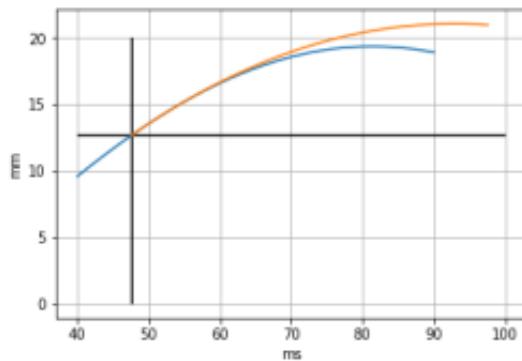
The general integral is  $x = Q + R \exp(-2\zeta\omega_n\tau) + S\tau$ ,  
 $v = -2\zeta\omega_n R \exp(-2\zeta\omega_n\tau) + S$  and  $x(0) = Q + R = x_p$ ,  
 $v(0) = -2\zeta\omega_n R + S = v_p$ .

```
In [15]: R = -(vp-S)/2/z/wn
Q = xp - R

Xp = lambda s: Q + R*exp(-2*z*wn*s) + S*s
Vp = lambda s: -2*z*wn*R*exp(-2*z*wn*s) + S
```

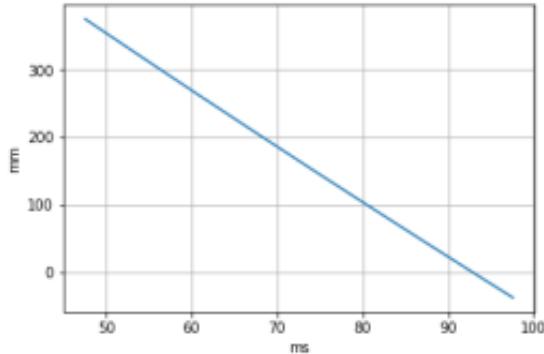
## Plotting the displacements, plastic branch

```
In [16]: plt.plot((t2+t0)*1000, 1000*f0(t2))      # elastic, blue
    plt.plot((t2+ty+t0)*1000, 1000*Xp(t2))    # plastic, red
    plt.hlines(xy*1000, 40, 100) ; plt.vlines((ty+t0)*1000, 0, 20)
    plt.xlabel('ms') ; plt.ylabel('mm')
    plt.grid();
```



### Plotting the velocity in plastic range

```
In [17]: plt.plot((t2+ty+t0)*1000, 1000*Vp(t2))  
plt.xlabel('ms') ; plt.ylabel('mm') ; plt.grid();
```



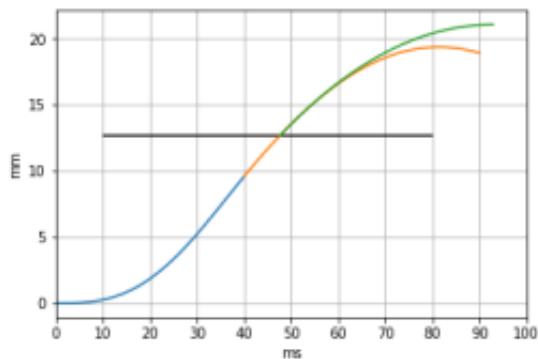
As you can see, the final velocity is negative, this means that our solution is invalid (we are in an elastic unloading branch).

**And the maximum displacement ( $v = 0$ ) is..**

```
In [18]: tVp0 = newton(Vp, 0.05)
print('Time of max x: ', 1000*(tVp0+ty+t0), 'ms,\nMax x: ', 1000*Xp(tVp0), ' mm.')
Time of max x: 92.87120039276594 ms,
Max x: 21.065084067663083 mm.
```

## Plotting it all together, forced, free, plastic

```
In [19]: tp = linspace(0, tVp0, 201)
plt.plot(1000*t, 1000*x) ; plt.plot(1000*(t2+t0), 1000*x2) # elastic
plt.plot(1000*(t0+ty+tp), 1000*Xp(tp)) # plastic
plt.xlim((0,100)) ; plt.xticks(arange(0,101,10))
plt.ylabel('mm') ; plt.xlabel('ms')
plt.hlines(xy*1000, 10, 80); plt.grid();
```



## Numerical Solution

### Discretize Time and Force

```
In [20]: Nsteps = 100
T = linspace(0, tVp0+ty+t0, Nsteps+1)
P = array([p0*sin(pi*t/t0) if t<t0 else 0 for t in T])
h = (tVp0+ty+t0)/Nsteps
```

## Constants for Constant Acceleration Algorithm

```
In [21]: DPm, DPc = 2*mass, 4*mass/h + 2*damp  
km, kd = 4*mass/h/h, 2*damp/h
```

## Initial Values, Initialize Storage to Save Our Results

```
In [22]: x0, v0 = 0.0, 0.0 ; X = [x0]
```

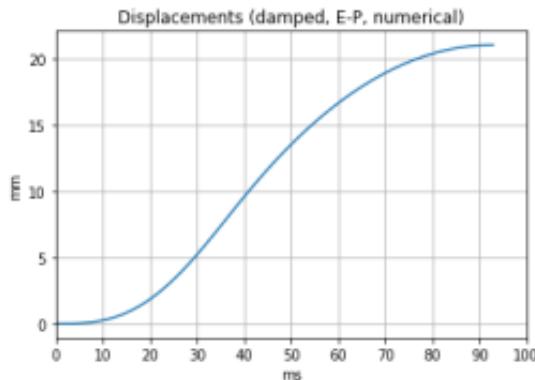
## Stepping

Simplified algorithm, no Modified Newton-Raphson

```
In [23]: for P0, DP in zip(P, P[1:]-P[:-1]):  
    a0=(P0-damp*v0-(stif*x0 if x0<xy else fy))/mass  
    DP = DP + DPc*v0 + DPM*a0  
    k_ = (stif if x0<xy else 0) + km + kd  
    dx = DP_/k_  
    # prepare for next step: update state variables, save X  
    x0, v0 = x0+dx, 2*dx/h - v0 ; X.append(x0)  
X = array(X)
```

## The never missing plot

```
In [24]: plt.plot(1000*T, 1000*X); plt.grid()
plt.ylabel('mm') ; plt.xlabel('ms')
plt.xticks(range(0,101,10)) ; plt.xlim((0,100))
plt.title('Displacements (damped, E-P, numerical)')
plt.show()
```



### Let's check the numerical result.

We print the numerical result, the exact result and the percentual of the difference.

```
In [25]: xmxn = 1000*max(X)
xmxn = 1000*Xp(tVp0)
print('Max displacement, numerical: ', xmxn, 'mm,')
print('Max displacement, exact:      ', xmxn, 'mm,')
print('(max_n-max_e)/max_e [%]:    ', '%f%%' % (100*(xmxn-xmxn)/xmxn))
```

```
Max displacement, numerical:  21.049417325626646 mm,
Max displacement, exact:      21.065084067663083 mm,
(max_n-max_e)/max_e [%]:    -0.074373%
```